

FUN3D v14.0 Training: A Few Attractive Additional Features for Internal Flow Simulations*

Jan Carlson
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*Features applicable to external flow simulations as well.



Today's Features

- Typical boundary conditions for internal flows
- Individual component performance tracking
- Back pressure controller
- Porous media modeling
- Three simple “turbofan” engine models



Sequence of Events

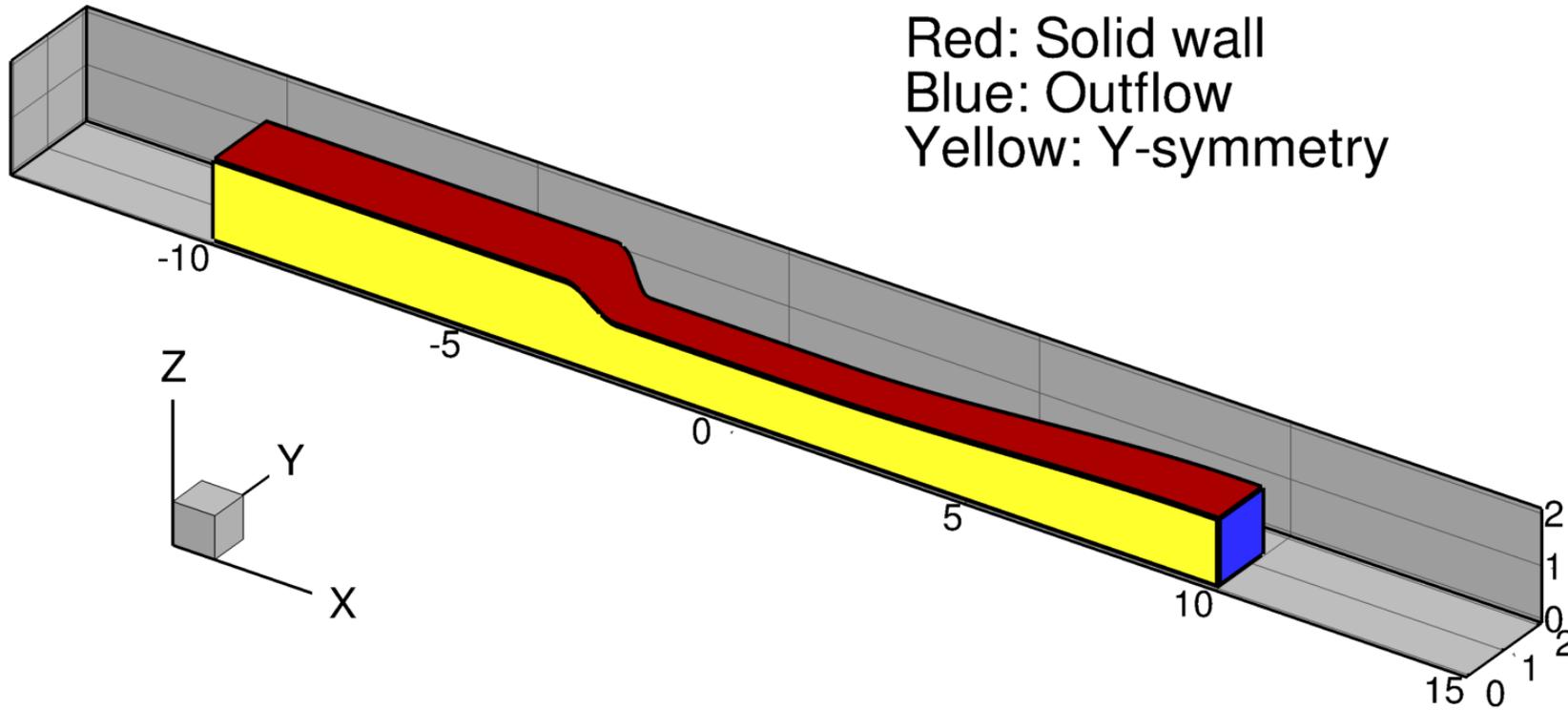
- Simple internal flow
 - Initial solution using fixed boundary conditions
 - Assessing solution development via component tracking
 - Adjusting flow conditions using a back pressure controller
 - Adding honeycomb/heat exchanger modeling using a porous media method
 - Readjusted flow conditions using the controller
- Simple engine model
 - Coupling boundary conditions



- External and mixed external/internal flows
 - Far-field freestream exists, but can be different from the reference
 - $M_\infty = M_{\text{ref}}$ (notable exception being many rotorcraft simulations)
 - $p_\infty = p_{\text{ref}} = 1/\gamma$
 - $p_{t\infty} = p_\infty \left(1 + \frac{1}{2} (\gamma - 1) M_\infty^2 \right)^{\gamma/\gamma-1}$
- Internal flows
 - No far-field freestream exists
 - Reference conditions *can* be different than “freestream”, but...
 - The pressure field may or may not be uniquely defined (depends upon the choice of boundary conditions)



Perspective View of Converging-Diverging Duct



Red: Solid wall
Blue: Outflow
Yellow: Y-symmetry

mapbc file

```
6
1 6663 z-symmetry
2 xxxx wall
3 xxxx inflow
4 xxxx outflow
5 6662 y-symmetry
6 xxxx wall
```

- 1/4-plane symmetric model
- Flow from left to right
- Boundary faces: 1, bottom; 2, top; 3, left; 4, right; 5: this side; 6: that side.



Boundary Condition Types

mapbc file*

```

6
1 6663 z-symmetry
2 4000 wall
3 7011 inflow
4 5051 outflow
5 6662 y-symmetry
6 4000 no-slip wall

```

No additional input required

3000: Tangency
 4000: Strong no slip
 4100: Wall function
 4110: Weak no slip

7011: Subsonic inflow, total conditions
 7036: Mass inflow
 7100: Fixed (supersonic)
 Velocity-Temperature (not yet released)

5051: Static pressure, supersonic extrapolate
 5052: Mach number
 5061: Static pressure, non-reflecting
 7012: Static pressure, subsonic, no reverse flow
 7031: Mass outflow
 7200: Solution functional

3000: Tangency
 4000: Strong no slip
 4100: Wall function
 4110: Weak no slip

No additional input required

*Not all combinations work or work well.



Terms and Conditions May Apply

- With no far-field boundaries, there is no “freestream”
- For most simulations, it is useful to define “freestream/infinity” as the condition existing in the area of interest, e.g., a test section or the area minimum, at any particular moment in time
- When using the total conditions—static pressure inflow-outflow combination for boundary types:
 - The values of the total pressure and temperature at the inflow are determined using the isentropic relations with the reference Mach number (thinking as an external flow), i.e.,
 - $p_t/p_{ref} = f(M_{ref})$
 - $T_t/T_{ref} = f(M_{ref})$
 - The freestream conditions can be modulated to compensate for viscous losses in the flow by using the back pressure at the outflow boundary to achieve $M_\infty = M_{ref}$



A complete listing of the fun3d.nml file



```
&project
project_rootname = 'baseline'
/
&governing_equations
  eqn_type      = 'compressible'
  viscous_terms = 'turbulent'
/
&turbulent_diffusion_models
  turbulence_model = 'sa-neg'
/

&raw_grid
  grid_format = 'aflr3'
/

&inviscid_flux_method
  flux_limiter      = 'none' ! default
  flux_construction = 'roe' ! default
  flux_construction_lhs = 'consistent' ! default is 'vanleer'
/

&nonlinear_solver_parameters
  schedule_iteration = 1 1000
  schedule_cfl       = 1. 50.
  schedule_cflturb   = 1. 20.
/

&code_run_control
  steps              = 25000
  restart_write_freq = 100
  stopping_tolerance = 1.0e-15
  restart_read       = 'off' ! 'on'
/
```



```
&global
  boundary_animation_freq = -1. ! output solution on boundaries at job end
/

&boundary_output_variables
  number_of_boundaries = -1 ! determine number of boundaries out from boundary_list
  boundary_list      = '1-6' ! list of boundaries to be included
  ptot              = T ! add these parameters
  htot              = T
  ttot              = T
  q_criterion       = T
  yplus             = T
/

&sampling_parameters
  number_of_geometries = 2
  type_of_geometry(1) = 'line' ! sample the solution between points p1_line and p2_line
    label(1) = 'centerline'
    p1_line(1:3,1) = -1000.0 0.01 0.01
    p2_line(1:3,1) = 1000.0 0.01 0.01
    variable_list(1) = 'x,y,z,rho,u,v,w,p,ptot,ttot,htot,mach' ! extract these parameters
  sampling_frequency(1) = -1 ! write file at end of run

  type_of_geometry(2) = 'volume_points' ! extract single points of data from the volume
  number_of_points(2) = 2
  sampling_frequency(2) = 1 ! sample every iteration/timestep
    points(:,2,1) = -0.5,0.01, 0.01 ! point 1 location
    points(:,2,2) = 0.5,0.01, 0.01 ! point 2 location
    variable_list(2) = 'p,ptot,u,mach' ! extract these parameters
    plot(2) = 'serial_history' ! format list as ASCII output list of data
/
```



```
&component_parameters      ! track F & M, as well as,
allow_flow_through_forces = T ! parameters such as mass flow, Mach number,
list_forces                = T ! pressure, and temperature
```

```
! track six components in this example, some are boundaries,
! some are slices through the mesh
number_of_components = 6
```

```
component_name(1) = 'inflow' ! component label
component_input(1) = '3'     ! inflow boundary boundary number(s)
component_count(1) = -1     ! extract count from component_input list
component_symmetry(1) = 4    ! quarter-plane symmetric grid
```

```
component_name(2) = 'outflow' ! component label
component_input(2) = '4'     ! outflow boundary number(s)
component_count(2) = 1      ! here the count is explicitly set
component_symmetry(2) = 4    ! quarter-plane symmetric grid
```

```
component_name(3) = 'total' ! combine inflow and outflow for mass flow
component_input(3) = '3,4'  ! imbalance as a convergence indicator
component_count(3) = -1
```

```
component_count(4) = 1      ! sample the middle of the test
component_input(4) = '0'    ! section for conditions
component_type(4) = 'circle' ! geometry cut will be a circle
circle_center(1:3,4) = 0.0, 0.0, 0.0 ! center of circle (x,y,z)
circle_normal(1:3,4) = 1.0, 0.0, 0.0 ! direction normal to plane of circle
circle_radius(4) = 3.0.    ! radius of circle
component_name(4) = 'throat' ! component label
component_symmetry(4) = 4    ! quarter-plane symmetry
```

```
component_count(5) = 1 ! sample at Sta. -9.
component_input(5) = '0' ! component not a boundary
component_type(5) = 'circle'
circle_center(1:3,5) = -9.0, 0.0, 0.0
circle_normal(1:3,5) = 1.0, 0.0, 0.0
circle_radius(5) = 10.0
component_name(5) = 'Sta.1' ! component label
component_symmetry(5) = 4
```

```
component_count(6) = 1 ! sample at Sta. -7.
component_input(6) = '0' ! component not a boundary
component_type(6) = 'circle'
circle_center(1:3,6) = -7.0, 0.0, 0.0
circle_normal(1:3,6) = 1.0, 0.0, 0.0
circle_radius(6) = 10.0
component_name(6) = 'Sta.2' ! component label
component_symmetry(6) = 4
```

```
/
```



```
&reference_physical_properties
mach_number    = 0.2    ! Reference condition
reynolds_number = 5.0e+6 ! 1/[m]
temperature    = 288.15 ! Reference static temperature
temperature_units = 'Kelvin'
/

&boundary_conditions
  grid_units = 'meters'
  wall_temperature(2) = -1. ! pseudo-adiabatic condition
  wall_temp_flag(2) = T
  wall_temperature(6) = -1. ! pseudo-adiabatic condition
  wall_temp_flag(6) = T

  total_pressure_ratio(3) = 1.02828 ! M = 0.2 using (p_t/p) isentropic equation
  total_temperature_ratio(3) = 1.00800 ! M = 0.2 using (T_t/T) isentropic equation
  static_pressure_ratio(4) = 1.010 ! first guess
! solution functional
  dynamic_boundary_conditions = .false. ! Set to true to engage the boundary condition controller
  boundary_functional_name(4) = 'back_pressure' ! name of boundary condition used by the controller
  field_point1(4,:) = 0.0, 0.01, 0.01 ! survey point
  update_frequency(4) = 500
/

&flow_initialization
  number_of_volumes = 1
  type_of_volume(1) = 'sphere' ! V-Parr type initialization
  center(1:3,1) = 0.0 0.0 0.0
  radius(1) = 50.0
  rho(1) = 1.00
  u(1) = 0.10
  c(1) = 1.0
/
```



Useful Files Output From This Simulation

Useful diagnostic files

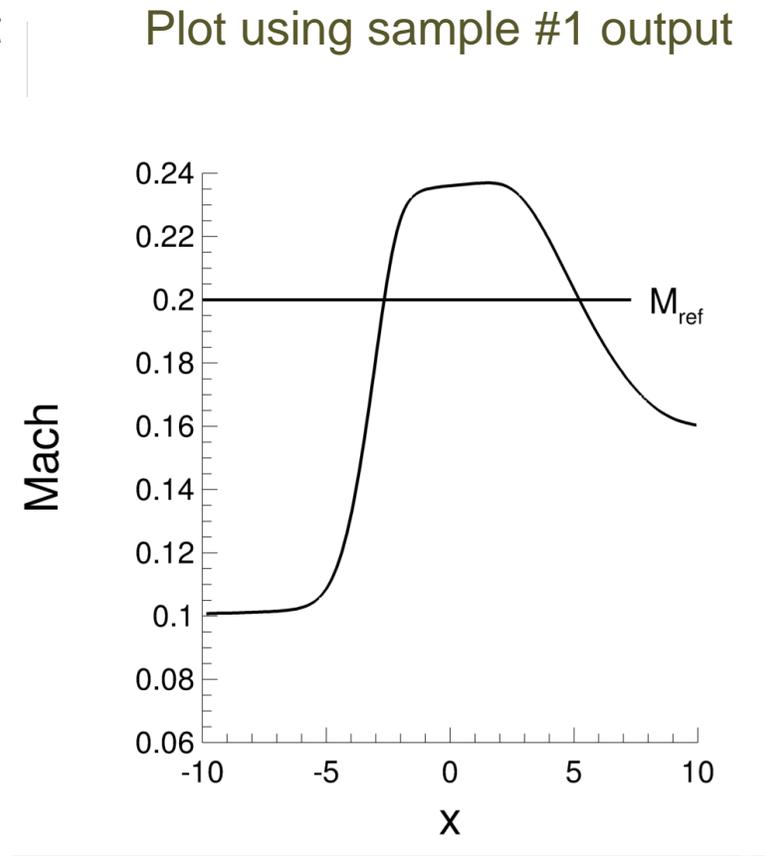
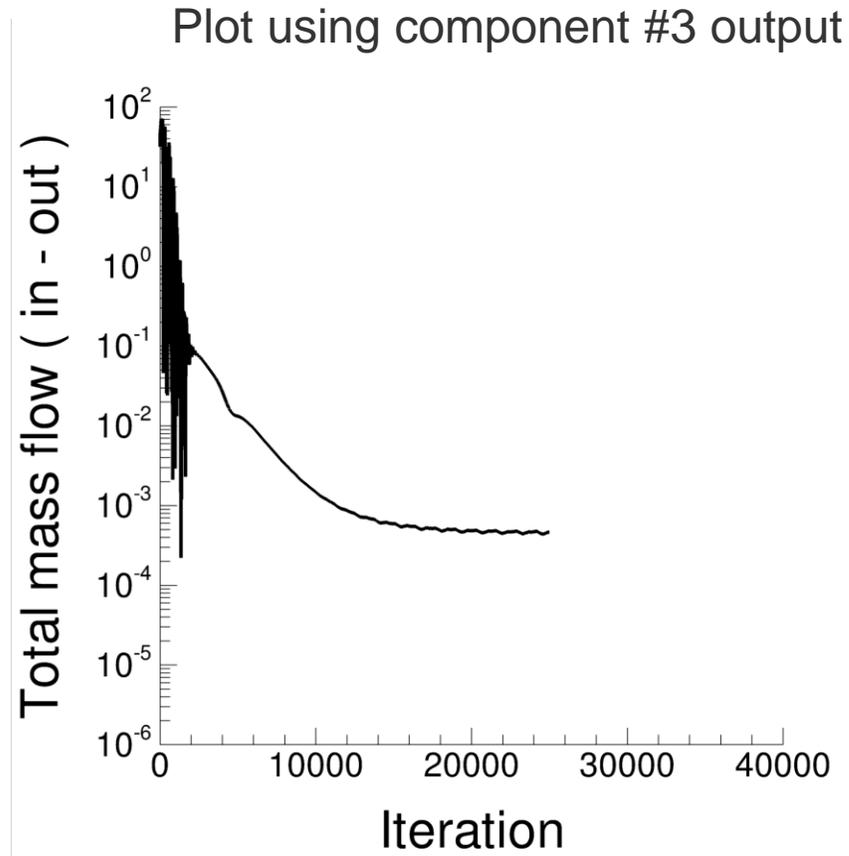
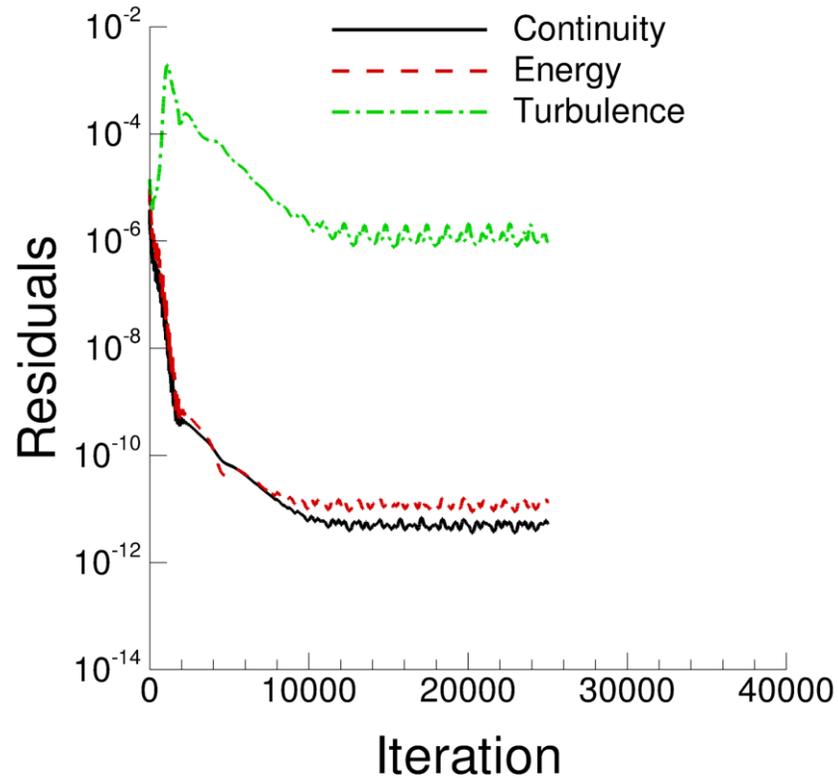
- [project_name]_hist.dat
 - Solution residuals
 - Simulation total, viscous, and pressure components of forces and moments
- [project_name]_fm_[component_name].dat
 - User defined component force and moment histories
 - “Flow through” area weighted statistics
 - Mass flow, density, velocity, Mach number,
 - Temperature, Total temperature
 - p/p_{ref} , p_{total}/p_{ref} ,
 - RMS temperature (generic gas path only)
 - X-momentum flux, Y-momentum flux, Z-momentum flux
- [project_name]_[boundary number]_controller.dat
 - History file of the PID-controller when the back pressure controller is engaged



- Plots
 - Residual history
 - Mass flow conservation
 - Centerline Mach number
- Components output



- First guess on back pressure, $p_{back} = p_{outflow}/p_{ref} = 1.010$, is a little low, pushing the centerline Mach number at $X = 0.0$ to close to $M_{\infty} = 0.24$. The desired condition is $M_{\infty} = M_{ref} = 0.20$.



Component performance summaries

FUN3D 14.0-8c4c24db0e Flow started 02/10/2023 at 09:25:56 with 400 processors.

 Component/stream information for project: baseline

Iteration = 25000
 Grid units = meters
 l_ref = 1.0000

Free stream:

Reynolds # = 5.0000E+06 per meters
 Mach (reference) = 0.200
 (farfield) = 0.2000 (Mach_reference * vinf_ratio)
 cv = 717.633 [J/kg-k]
 cp = 1004.686 [J/kg-k]
 gamma = 1.400
 Molecular weight = 28.9645 [kg/kmol]
 R (specific) = 287.0530 [J/kg-K]
 Equation type: cal_perf_compress
 Viscous terms: turbulent
 Chemical kinetics: not applicable
 Thermal energy: not applicable
 T_infinity = 288.1 [K]
 = 518.7 [R]
 = 59.0 [F]
 temperature_ref = 288.1 [K]
 r_infinity = 1.3146 [kg/m^3]
 = 0.82068E-01 [lbm/ft^3]
 = 0.25507E-02 [slug/ft^3]
 rho_ref = 0.0000 [kg/m^3]
 p_infinity = 108736.107 [Pa]
 = 15.770839 [lbf/sq.in.]
 = 2271.00081 [lbf/sq.ft.]

a_infinity = 340.29 [m/s]
 = 1116.45 [ft/s]
 = 761.22 [miles/hour]
 u_infinity = 68.059 [m/s]
 = 223.290 [ft/s]
 = 152.243 [miles/hour]
 v_ref = 0.00 [m/s]
 q_infinity = 3044.611 [Pa]
 = 0.442 [lbf/sq.in.]
 = 63.588 [lbf/sq.ft.]
 rho*a^2_infinity = 152230.550 [Pa]
 = 22.079175 [lbf/sq.in.]
 = 3179.40114 [lbf/sq.ft.]
 p,t_infinity = 111811.3 [Pa]
 = 16.217 [lbf/sq.in.]
 = 2335.227 [lbf/sq.ft.]
 = 1.0283 [Fun3d]
 T,t_infinity = 290.5 [K]
 = 522.8 [R]
 = 63.1 [F]
 bulk viscosity = 1.78940E-05 [kg/m-sec]
 = 1.0000 [non-dimensional]
 Time step (dt) = 0.0000 [non-dimensional]
 (dt/a_inf) = 0.0000 [s]
 Reference areas :
 aerodynamic = 1.0000 mesh^2, 1.0000 [m^2]
 propulsion = 1.0000 mesh^2, 1.0000 [m^2]

 Summary of plenum gas properties
 Boundary T_total Avg. Mol. wt R_specific gamma species

```

.....
inflow

component_type ( 1) = boundary
component_symmetry ( 1) = 4.0
shape direction ( 1) = 0.000 0.000 0.000
shape area = 2.25000E+00 mesh^2, 2.25000E+00 [m^2], 2.42188E+01 [ft^2]
shape area * symmetry = 9.00000E+00 mesh^2, 9.00000E+00 [m^2], 9.68752E+01 [ft^2]
average density = 1.015 [Fun3d], 1.3343 [kg/m^3], 0.08330 [lbm/ft^3]
average u velocity = 0.101 [Fun3d], 34.3469 [m/s], 112.687 [ft/s]
average v velocity = -0.000 [Fun3d], -0.0149 [m/s], -0.049 [ft/s]
average w velocity = -0.000 [Fun3d], -0.0149 [m/s], -0.049 [ft/s]
average static pressure = 0.729 [Fun3d], 111021.870 [Pa], 16.102 [psi]
velocity magnitude = 0.101 [Fun3d], 34.3469 [m/s], 112.687 [ft/s]
local speed of sound = 1.003 [Fun3d], 341.3070 [m/s], 1119.774 [ft/s]
local Mach number = 0.101
average total pressure = 0.734 [Fun3d], 111811.079 [Pa], 16.217 [psi]
minimum total pressure = 0.729 [Fun3d], 111026.497 [Pa], 16.103 [psi]
maximum total pressure = 0.735 [Fun3d], 111931.415 [Pa], 16.234 [psi]
average cp * Tt = 2.5200 [Fun3d], 2.91816E+05 [J/kg]
average cp * T = 2.5149 [Fun3d], 2.91226E+05 [J/kg]
average entropy = 0.34074E-06 [Fun3d], 0.13693E-03 [J/K]
average static temperature = 1.006 [Fun3d], 289.868 [K], 521.762 [R]
average total temperature = 1.008 [Fun3d], 290.455 [K], 522.819 [R]
-----
mass flow = 0.92200 [Fun3d], 412.45502 [kg/s], 909.30767 [lbm/s]
mass flow ( corrected ) = 0.83886 [Fun3d], 375.26552 [kg/s], 827.31886 [lbm/s]
: sqrt(T_total / T_std.) = 1.004
: p_total / p_std. = 1.103
-----

```

Flow-through

X-momentum flux = 0.93081E-01 [Fun3D], 0.142E+05 [N], 3185.5 [lbf]
 Y-momentum flux = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf]
 Z-momentum flux = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf]
 Delta X-pressure flux = 0.13514 [Fun3D], 0.206E+05 [N], 4624.7 [lbf]
 Delta Y-pressure flux = 0.0000 [Fun3D], 0.00 [N], 0.0000 [lbf]
 Delta Z-pressure flux = 0.0000 [Fun3D], 0.00 [N], 0.0000 [lbf]

Totals

X-Viscous forces = 0.93081E-01 [Fun3D], 0.142E+05 [N], 3185.5 [lbf] 4.654 [1/Q infinity]
 Y-Viscous forces = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf] -0.002 [1/Q infinity]
 Z-Viscous forces = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf] -0.002 [1/Q infinity]
 Delta X-Pressure forces = 0.13514 [Fun3D], 0.206E+05 [N], 4624.7 [lbf] 6.757 [1/Q infinity]
 Delta Y-Pressure forces = 0.0000 [Fun3D], 0.00 [N], 0.0000 [lbf] 0.000 [1/Q infinity]
 Delta Z-Pressure forces = 0.0000 [Fun3D], 0.00 [N], 0.0000 [lbf] 0.000 [1/Q infinity]
 X-vacuum forces = 0.33581E-04 [Fun3D], 5.11 [N], 1.1492 [lbf]
 Y-vacuum forces = 0.0000 [Fun3D], 0.00 [N], 0.0000 [lbf]
 Z-vacuum forces = 0.0000 [Fun3D], 0.00 [N], 0.0000 [lbf]

X-Total forces = 0.22822 [Fun3D], 0.347E+05 [N], 7810.2 [lbf] 11.411 [1/Q infinity]
 Y-Total forces = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf] -0.002 [1/Q infinity]
 Z-Total forces = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf] -0.002 [1/Q infinity]
 atan (F_y / F_x) = -0.10117E-01[deg.]
 atan (F_z / F_x) = -0.10117E-01[deg.]

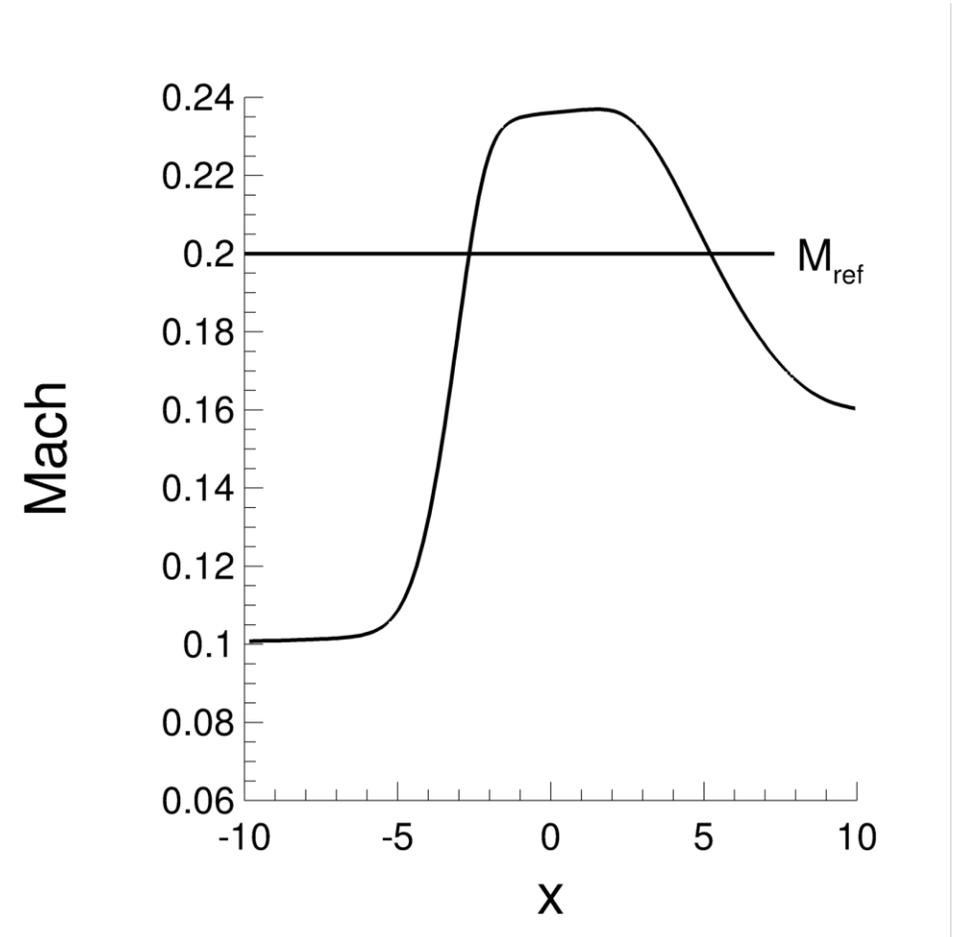
Rotated X-Total forces = 0.22822 [Fun3D], 0.347E+05 [N], 7810.2 [lbf] 11.411 [1/Q infinity]
 Rotated Y-Total forces = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf] -0.002 [1/Q infinity]
 Rotated Z-Total forces = -0.40299E-04 [Fun3D], -6.13 [N], -1.3791 [lbf] -0.002 [1/Q infinity]
 atan (F_y,rot / F_x,rot) = -0.10117E-01[deg.]
 atan (F_z,rot / F_x,rot) = -0.10117E-01[deg.]

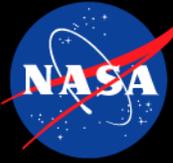
Total force = 0.22822 [Fun3D], 0.347E+05 [N], 7810.2 [lbf] 11.411 [1/Q infinity]
 Total vacuum force = 0.93115E-01 [Fun3D], 0.142E+05 [N], 3186.6 [lbf] 4.656 [1/Q infinity]



Modulate The Back Pressure

- The freestream Mach number is high due to a low value of the back pressure, $p_{\text{back}} = p_{\text{outflow}}/p_{\text{ref}} = 1.010$, being a bit low
- Tactics to bring M_{∞} down to M_{ref}
 - Manual trial and error
 - This often is the more expedient course of action for simulations with long or unknown response times
 - PID-control process
 - Takes some trial and error to determine the controller coefficients for new problems
 - Allows for automation of a series of simulations





Engage The Controller: update #1 fun3d.nml

Set dynamic_boundary_conditions to .true. in the fun3d.nml file

```

&boundary_conditions
      grid_units = 'meters'
      wall_temperature(2) = -1. ! psuedo-adiabatic condition
      wall_temp_flag(2) = T
      wall_temperature(6) = -1. ! psuedo-adiabatic condition
      wall_temp_flag(6) = T

      total_pressure_ratio(3) = 1.02828 ! M = 0.2 from (p/p_t) isentropic equation
      total_temperature_ratio(3) = 1.00800 ! M = 0.2 from (T/T_t) isentropic equation
      static_pressure_ratio(4) = 1.010 ! first guess
! solution functional
      dynamic_boundary_conditions = .true. ! Set to true to engage the back pressure controller
      boundary_functional_name(4) = 'back_pressure'
      field_point1(4,:) = 0.0, 0.01, 0.01 ! survey point
      update_frequency(4) = 500
/

```

Edit the [project_name].mapbc file

mapbc file

```

6
1 6663 z-symmetry
2 4000 wall
3 7011 inflow
4 7200 outflow solution functional
5 6662 y-symmetry
6 4000 no-slip wall

```

Create a new, separate namelist file: controller.nml

```

&tunnel_control
  number_of_controllers = 1 ! one controller to set up
  controller_points(4) = 1 ! control boundary 4
      find_mach(4) = .true. ! objective is Mach number
      target_mach(4) = 0.20. ! Mach number value
      initial_delay(4) = 1000 ! to allow transients to die down
          kp(4) = 0.025 ! proportional
          ki(4) = 0.2e-5 ! integral
          kd(4) = 1.0e-6 ! derivative
/

```

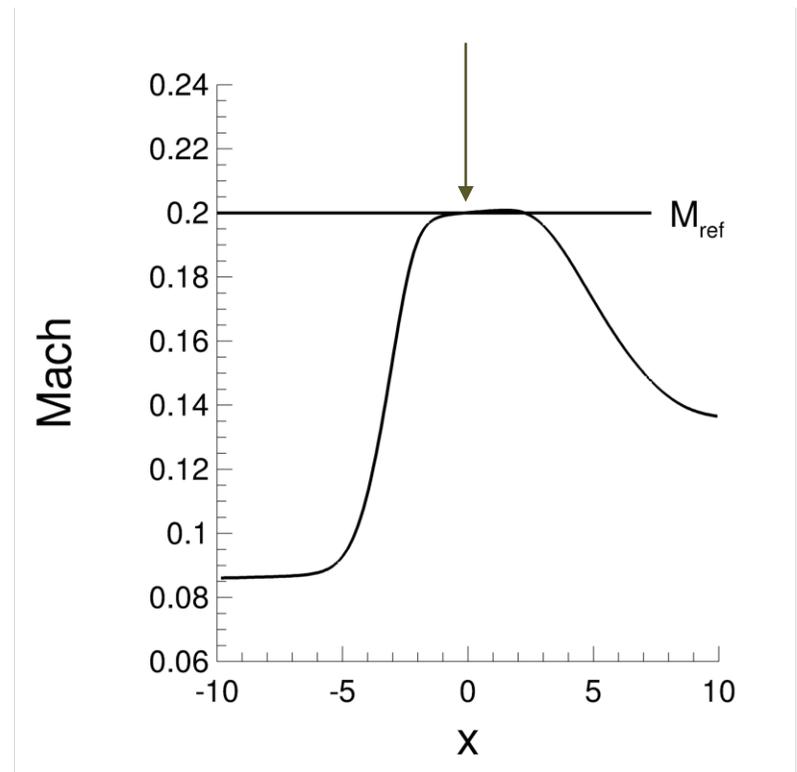
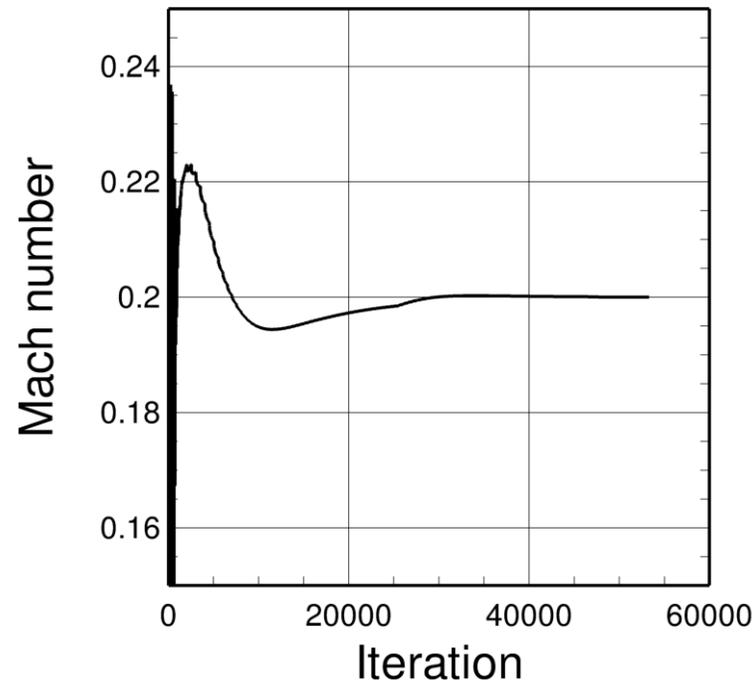
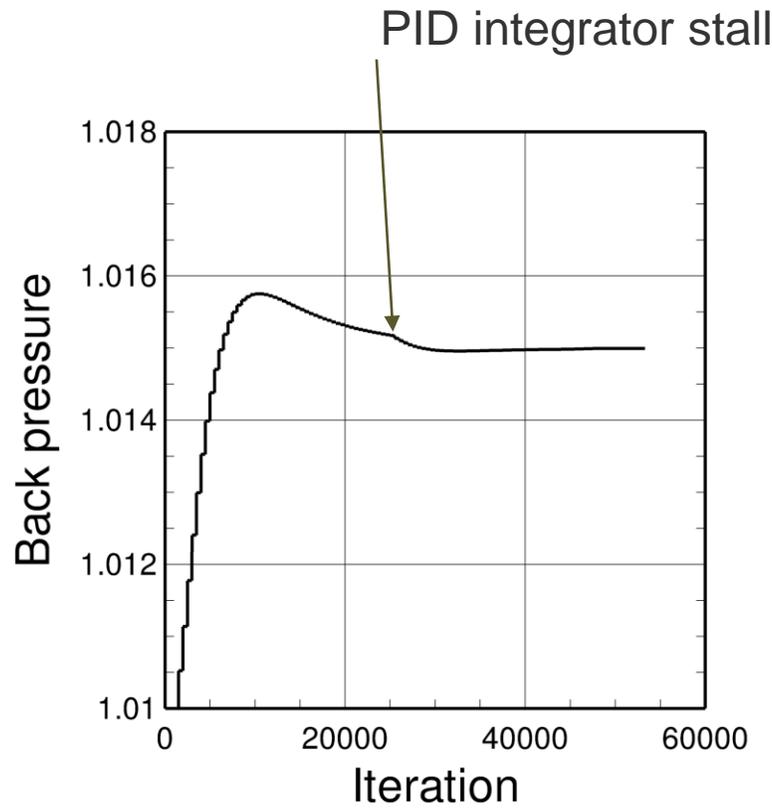


- Back pressure history
- Test section Mach number history
- Centerline Mach number



Back Pressure Controller

- Back pressure and Mach number output from [project_name]_[boundary]_controller.dat
- Centerline plot using the same sampling output file as previously discussed





Example of Porous Media Modeling

- Add a screen/honeycomb model in the upstream section
- Keep the back pressure controller active
- Examine back pressure history and centerline Mach number
- Plot centerline total pressure



Honeycomb Model: update #2 fun3d.nml

- Add a model of a honeycomb in the upstream portion of the tunnel
- &filters is part of the fun3d.nml file

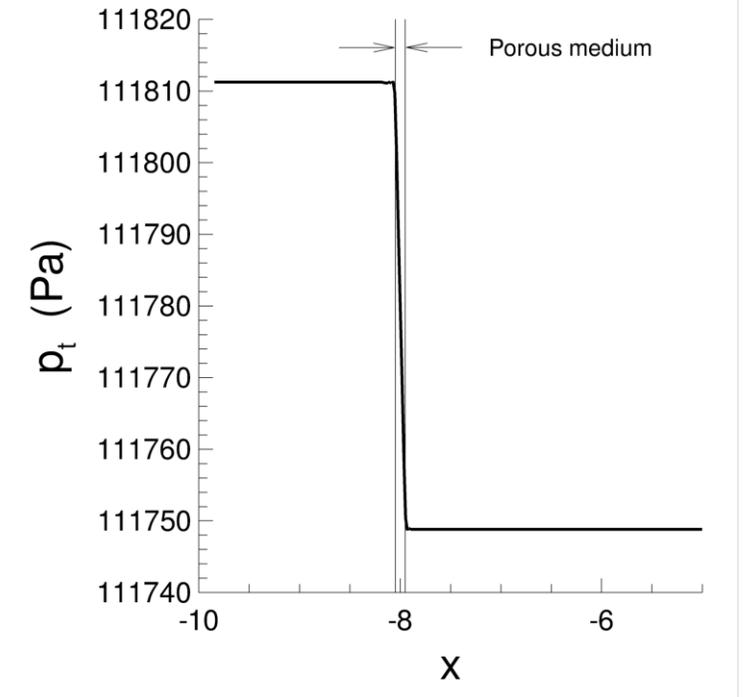
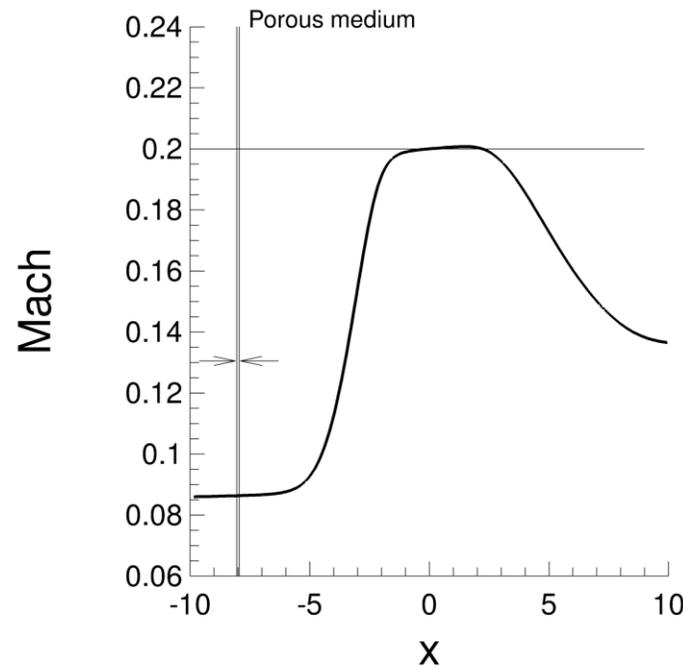
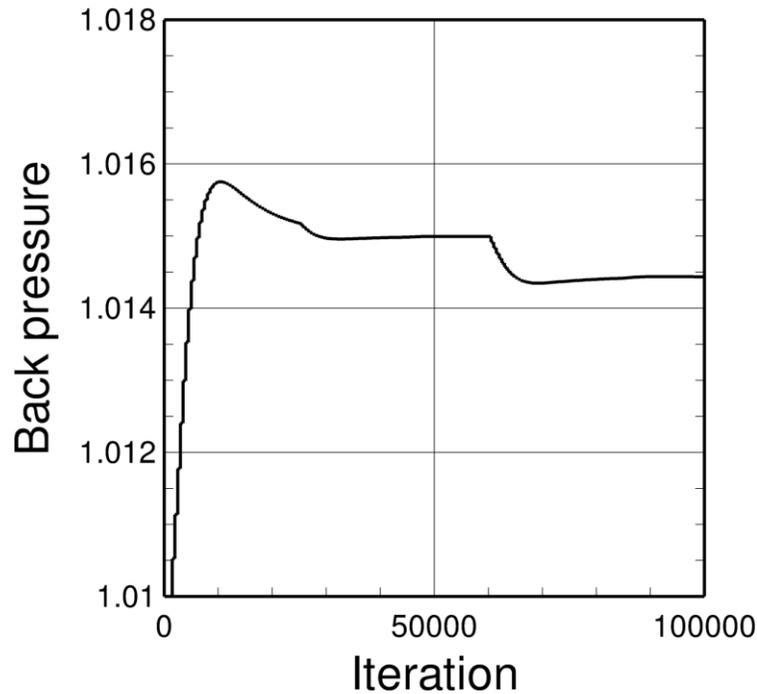
```
&filters
  passive_filter_flag = T ! activate porous media model
  number_of_fences   = 1 ! list fences (volumetric) first
  gain(1)            = 1.0 ! default
  permeability(1)    = 1.0E+35 ! essentially no viscous part
  fence_thickness(1) = 0.1 ! meters
  ! pressure loss factor (continuity, x-mom, y-mom, z-mom)
  pressure_loss_factor(1,:) = 0.0 1.0 0. 0. ! [1/m]
  fence_shape(1)      = 'hex' !
  corners(1,1,:) = -8.05 0.0 0.00
  corners(1,2,:) = -7.95 0.0 0.00
  corners(1,3,:) = -7.95 1.50 0.00
  corners(1,4,:) = -8.05 1.50 0.00
  corners(1,5,:) = -8.05 0.0 1.50
  corners(1,6,:) = -7.95 0.0 1.50
  corners(1,7,:) = -7.95 1.50 1.50
  corners(1,8,:) = -8.05 1.50 1.50
```

/



Restarting From Previous Solution

- To maintain the requested centerline Mach number, the back pressure must shift to compensate for the total pressure loss incurred by porous medium (honeycomb) model
- In this example, approximately 60 Pa total pressure drop simulated. The pressure loss is a function of the local dynamic pressure, $\Delta p_t / \Delta x = C q_{local}$

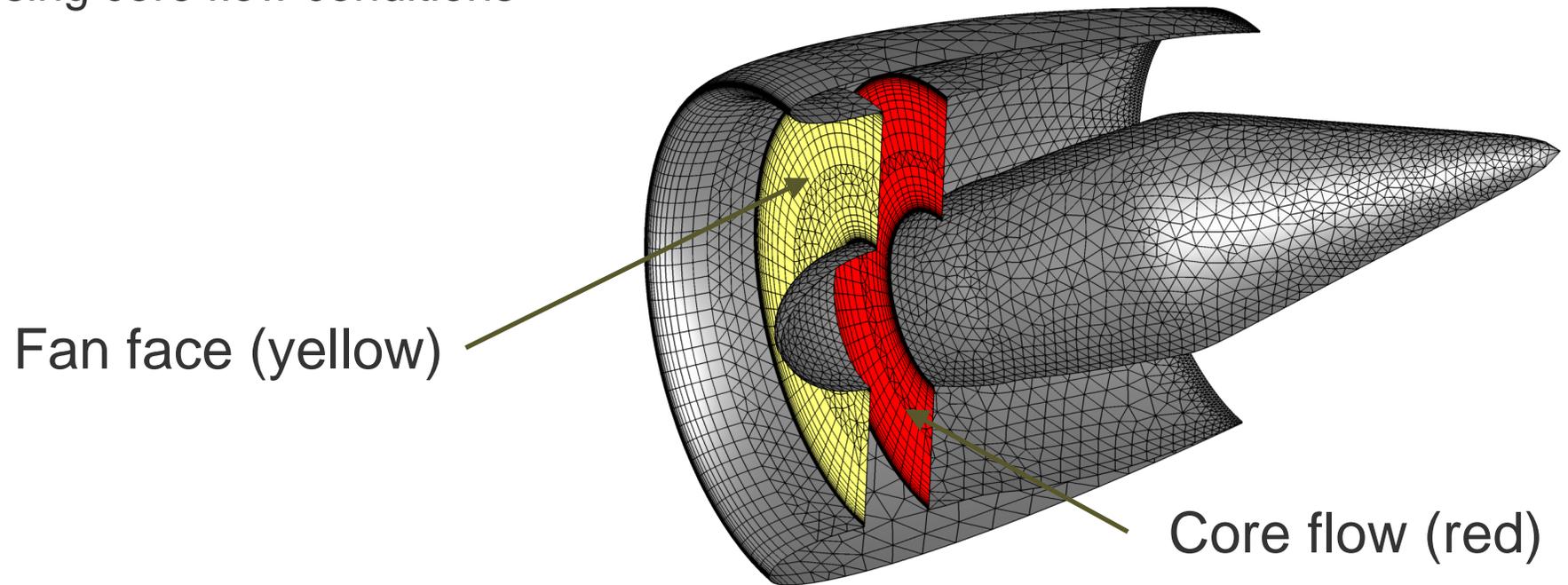
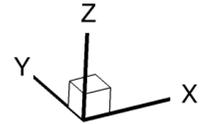




Three Simple Engine Simulations

Matching mass flow rates for a simple single stream engine

1. Set mass flow rates separately
2. Drive core flow using the fan face mass flow rate
3. Drive fan face flow using core flow conditions





Engine 1: Set mass flow rates separately

- Fan – boundary condition number 7031
- Core – boundary condition number 7036
- When using these boundary conditions, the input should be in units of grid units squared.
- From the reference conditions: $\rho_{\text{ref}} c_{\text{ref}} = 106.84 \text{ kg/s-m}^2$
- Suppose the target mass flow rate is $\dot{m}_{\text{target}} = 2.183 \text{ kg/s}$
- To convert to FUN3D units: $\dot{m}_{\text{target}} / \rho_{\text{ref}} c_{\text{ref}} = 2.0435\text{e-}2 \text{ 1/m}^2$
 $= 31.675 \text{ 1/inch}^2$
- Configure boundary condition namelist

```
&boundary_conditions
  grid_units = 'inches'
  massflow(1) = 31.675 ! core
  massflow(2) = 31.675 ! fan
  wall_temp_flag(3) = .true. ! no-slip
  wall_temperature(3) = -1.0
/
```



fun3d.nml for Engine 1

```
&global
volume_animation_freq = -1
boundary_animation_freq = -1
/

&reference_physical_properties
mach_number = 0.72
reynolds_number = 136721 ! 1/[in]
temperature = 392.4
temperature_units = 'Rankine'
/

&governing_equations
viscous_terms = 'turbulent'
/

&turbulent_diffusion_models
turbulence_model = 'sa-neg'
reynolds_stress_model = 'qcr2020'
use_diff_element = T
/

&spalart
turbinf = 3.0
/

&debug
weighted_lsq_diffusion_m = .true.
weighted_lsq_diffusion_t = .true.
/

&code_run_control
steps = 50000
restart_read = 'off'
stopping_tolerance = 1.0e-15
/

&linear_solver_parameters
linear_projection = .false.
/

&nonlinear_solver_parameters
time_accuracy = 'steady'
schedule_iteration = 1 250
schedule_cfl = 1. 50.
schedule_cfturb = 1. 20.
/

&inviscid_flux_method
flux_limiter = 'none'
flux_construction = 'roe'
/

&boundary_conditions
grid_units = 'inches'
massflow(1) = 31.675 ! core
massflow(2) = 31.675 ! fan
wall_temp_flag(3) = .true. ! no-slip
wall_temperature(3) = -1.0
/

&component_parameters

allow_flow_through_forces = T
number_of_components = 5

component_count(1) = 1
component_input(1) = '2'
component_name(1) = 'inlet'
component_symmetry(1) = 2.
```



fun3d.nml for Engine 1

```
component_count(2) = 1
component_input(2) = '1'
component_name(2) = 'exit'
component_symmetry(2) = 2.

component_count(3) = 2
component_input(3) = '1,2'
component_name(3) = 'balance'
component_symmetry(3) = 2.

component_count(4) = 3
component_input(4) = '1,2,3'
component_name(4) = 'total'
component_symmetry(4) = 2.

component_count(5) = 1
component_input(5) = '3'
component_name(5) = 'nacelle'
component_symmetry(5) = 2.
/

&volume_output_variables
export_to      = 'tec'
x              = .true.
y              = .true.
z              = .true.
primitive_variables = .true.
mach           = .true.
ptot           = .true.
ttot           = .true.
/

&boundary_output_variables
number_of_boundaries = 5
boundary_list = '1-5'
primitive_variables = .true.
mach = .true.
ttot = .true.
ptot = .true.
yplus = .true.
uavg = .true.
vavg = .true.
wavg = .true.
/

&project
project_rootname = 'engine_only'
/

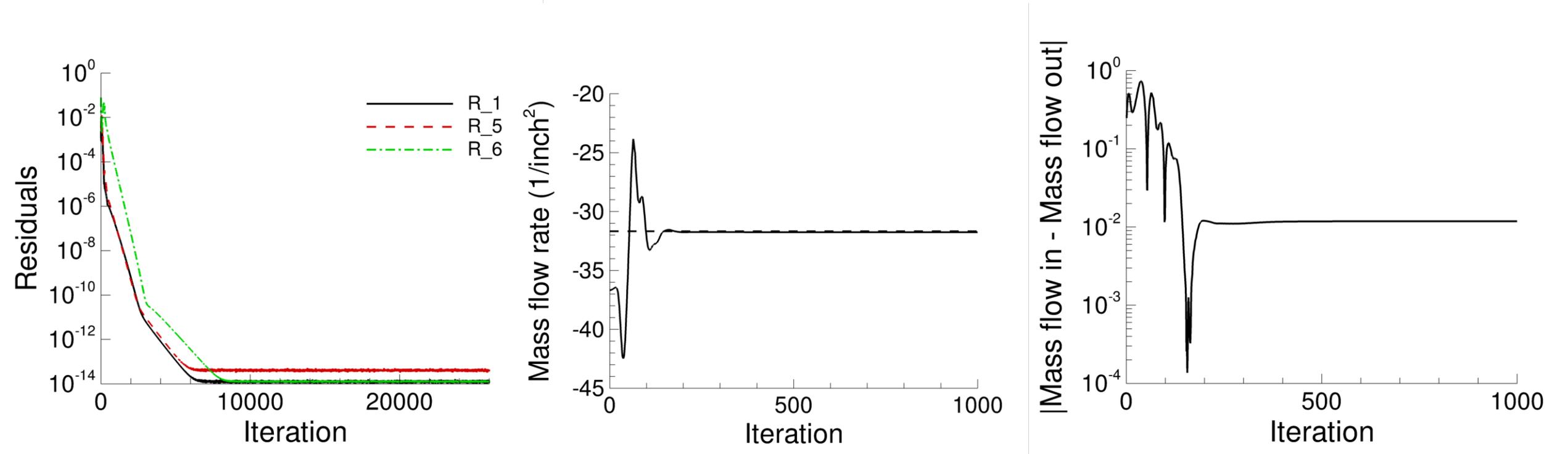
&raw_grid
grid_format = 'aflr3'
data_format = 'stream'
patch_lumping = 'family'
/

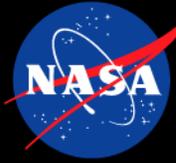
&force_moment_integ_properties
area_reference = 95.0
x_moment_center = 0.0
y_moment_center = 0.0
z_moment_center = 0.0
/
```



Engine 1: Solution convergence

- Iterative convergence
- Mass flow rate achieved $\approx 0.2\%$
- Inlet and core flow balanced $\approx 0.6\%$





Engine 2a: Drive core flow via the fan face flow

- Fan – boundary condition number 7031
- Core – boundary condition number 7036
- Fan boundary condition the same as Engine1
- No need to set core boundary condition value
- Configure mass flow rate matching

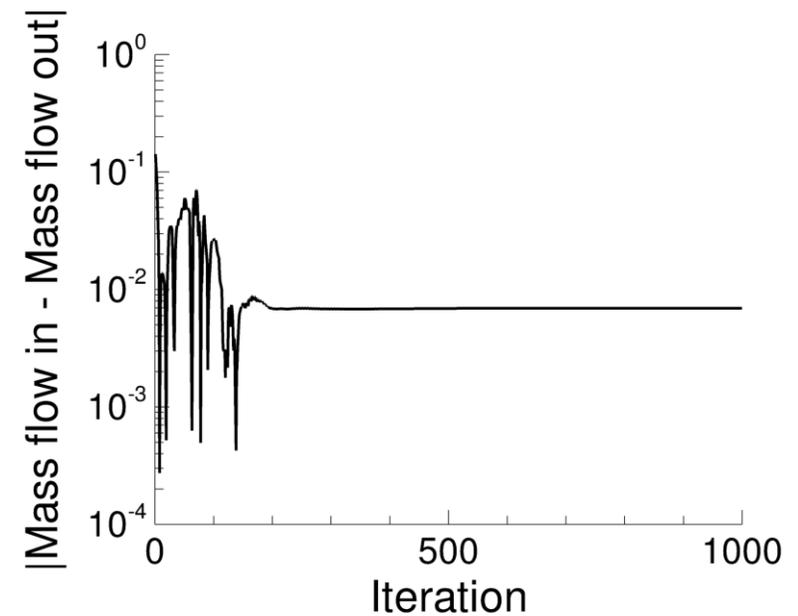
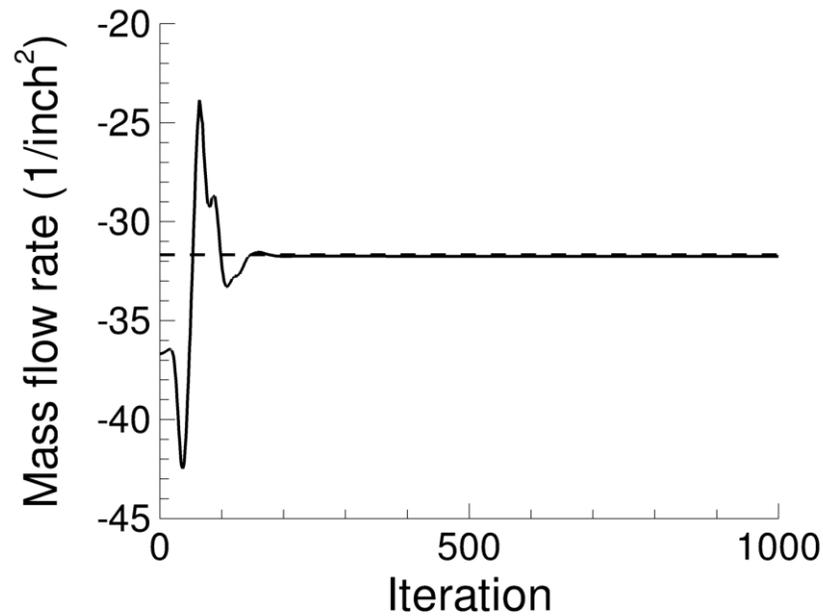
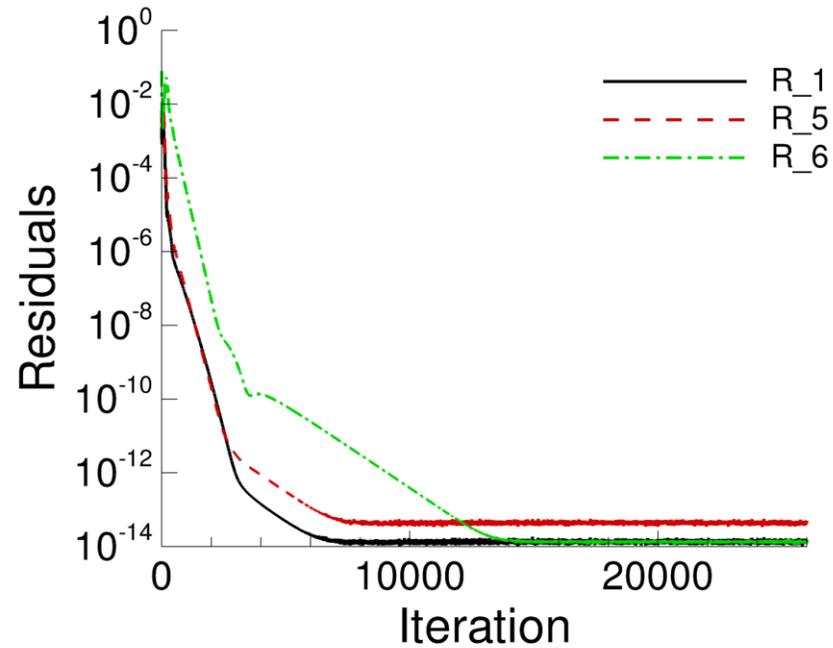
```
&boundary_conditions
  grid_units = 'inches'
  massflow(2) = 31.675 ! fan
  wall_temp_flag(3) = .true. ! no-slip
  wall_temperature(3) = -1.0

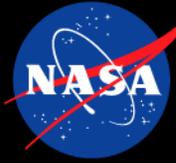
  number_of_engines = 1 ! just one engine here
  engine_symmetry(1) = 2.0 ! grid is half plane symmetric
  number_of_streams(1) = 1 ! single stream engine
  inlet_bc(1) = 2 ! fan bc – driver
  core_bc(1) = 1 ! core bc - follower
  cycle_name(1) = 'massflow' ! mass flow bc
/
```



Engine 2a: Solution convergence

- Iterative convergence
- Mass flow rate achieved $\approx 0.2\%$
- Inlet and core flow balanced $\approx 0.3\%$





Engine 2b: Drive core flow via the fan face flow

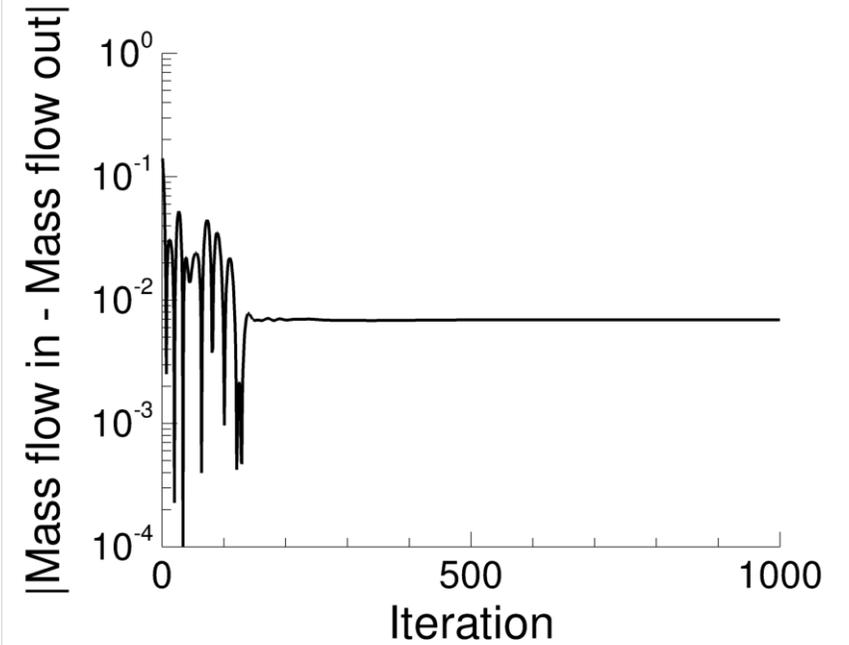
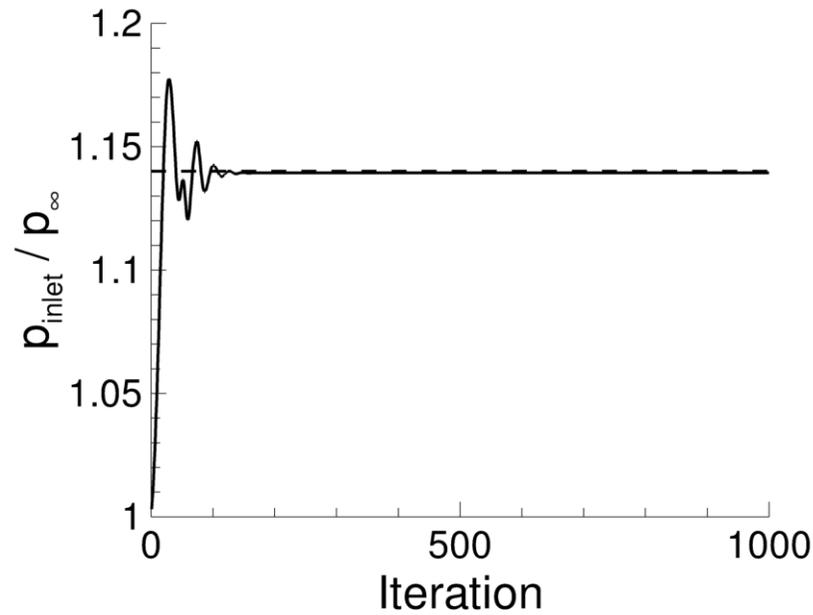
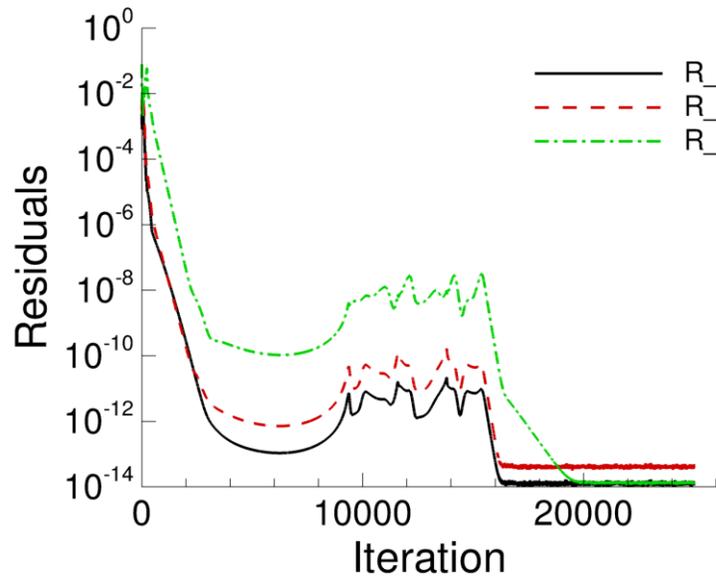
- Fan – boundary condition number 7012
- Core – boundary condition number 7036
- Fan boundary condition the same as Engine1
- No need to set core boundary condition value
- Configure mass flow rate matching

```
&boundary_conditions
  grid_units = 'inches'
  static_pressure_ratio(2) = 1.140 ! fan
  wall_temp_flag(3) = .true. ! no-slip
  wall_temperature(3) = -1.0

  number_of_engines = 1 ! just one engine here
  engine_symmetry(1) = 2.0 ! grid is half plane symmetric
  number_of_streams(1) = 1 ! single stream engine
  inlet_bc(1) = 2 ! fan bc – driver
  core_bc(1) = 1 ! core bc - follower
  cycle_name(1) = 'massflow' ! mass flow bc
/
```

Engine 2b: Solution convergence

- Iterative convergence
- Static pressure ratio achieved $\approx 0.05\%$
- Inlet and core flow balanced $\approx 0.3\%$





Engine 3: Drive fan flow via the core flow

- Fan – boundary condition number 7031
- Core – boundary condition number 7011
- Set the core total pressure and temperature
- Configure mass flow rate matching

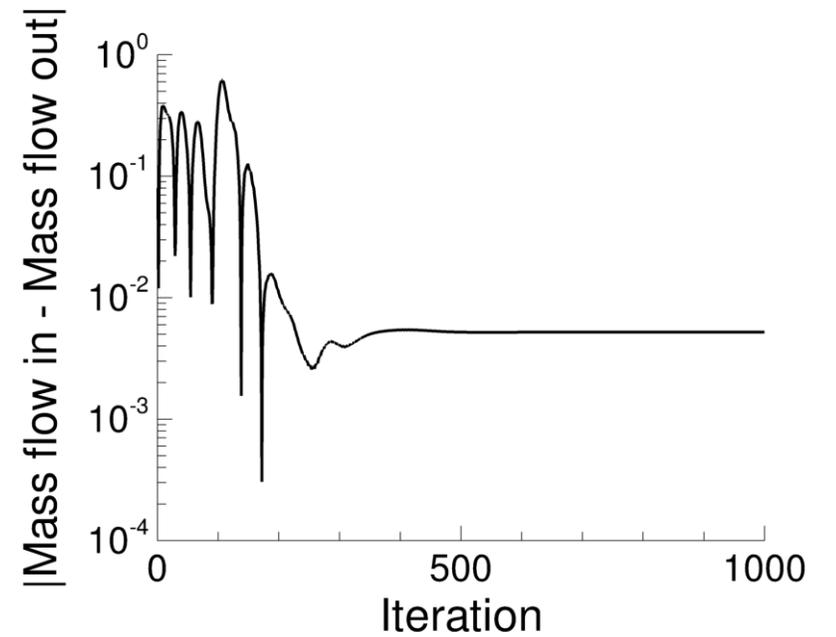
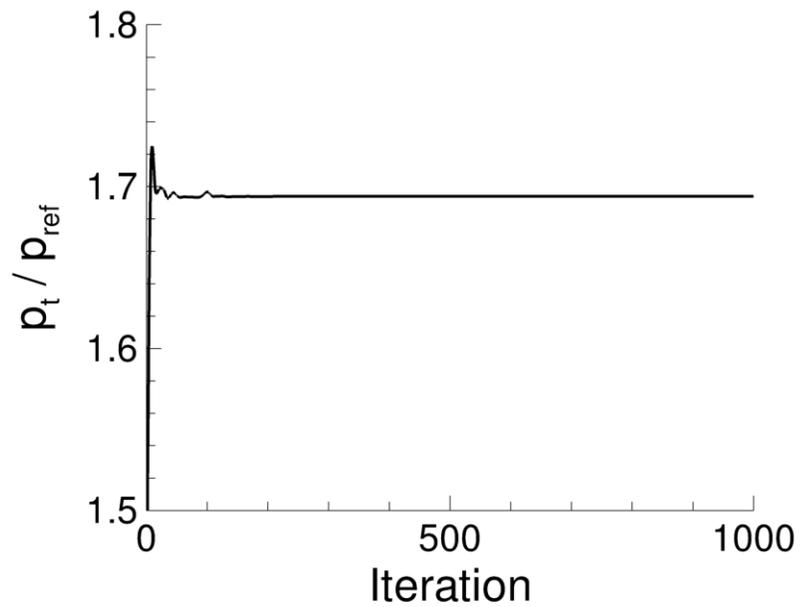
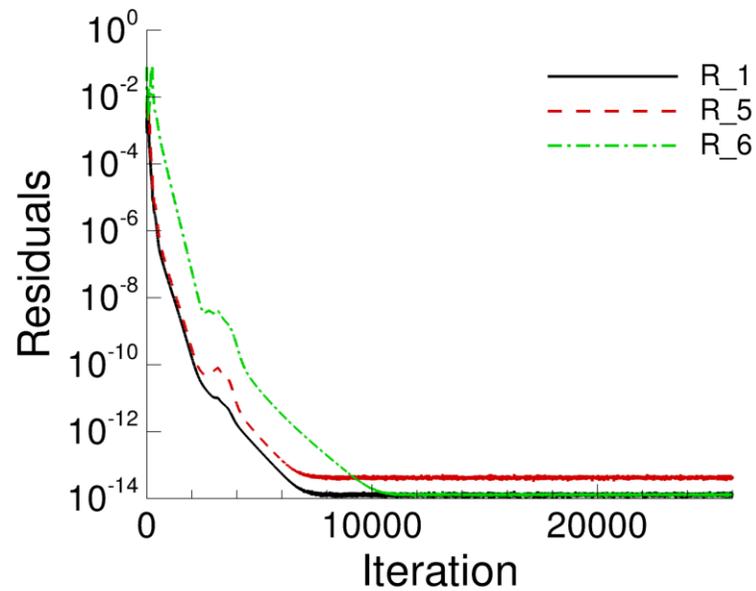
```
&boundary_conditions
  grid_units = 'inches'
  total_pressure_ratio(1) = 1.6949 ! core
  total_temperature_ratio(1) = 1.1627 ! core
  massflow(2) = 30.0 ! initial guess
  wall_temp_flag(3) = .true. ! no-slip
  wall_temperature(3) = -1.0

number_of_engines = 1 ! just one engine here
engine_symmetry(1) = 2.0 ! grid is half plane symmetric
number_of_streams(1) = 1 ! single stream engine
  inlet_bc(1) = 2 ! fan bc – driver
  core_bc(1) = 1 ! core bc – follower
  cycle_name(1) = 'total_conditions'
/
```



Engine 3: Solution convergence

- Iterative convergence
- Total pressure ratio achieved $\approx 0.05\%$
- Inlet and core flow balanced $\approx 0.3\%$





Today's Features Review

- Sketched out of some boundary conditions that can be used for internal flows
- F & M and flow tracking using component parameters
- An example using the back pressure controller
- An example of using the porous media model
- Discussed three simple “turbofan” engine models



Short List of References

- “Inflow/Outflow Boundary Conditions with Application to FUN3D,” NASA-TM-2011-208444
- “Boundary Condition Study for the Juncture Flow Experiment in the NASA Langley 14x22-Foot Subsonic Wind Tunnel,” AIAA-2017-4126
- “FUN3D and USM3D Analysis of the Propulsion Aerodynamic Workshop 2018 S-duct Test Case,” AIAA-2019-3848
- “Setting Boundary Conditions For Slotted Throat Wind Tunnels Using Calorically Imperfect Gas Assumptions,” AIAA-2022-0807